Research of New Machining Method of Skew Bevel Gears Based on Generation Line

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Abstract—According to generating principle of spiral bevel gear and space meshing theory, formation theory of generation line of tooth surface about skew bevel gears is studied. A new processing method of tooth surface with the generation line as chord of face circular cutters is put forward. Moving process model of milling processing with skew bevel gears is built, and profile of the gears can be machined using three axes linkage method based on the model. Machining test of the gears is carried out on experiment platform designed. Furthermore, 3D points cloud model of tooth surface are calculated in light of mathematical model of the tooth surface which has been established, and datum of mesh nodes on actual tooth surface are collected. Then surface errors is evaluated by normal vector equation of the mesh nodes on actual tooth surface which has been developed. Finally, this method is proved to be effective and feasible.

Index Terms—skew bevel gears, generation line, mathematical model of tooth surface, machining experiments

I. INTRODUCTION

Spiral bevel gears are needed to transfer power and motion between intersecting axes and are widely used in the equipments of aviation, navigation and national defense and mechanical products such as vehicles, machine tools, engineering mechanics[1]. Skew bevel gears are as a part of spiral bevel gears and have all advantages of spiral bevel gears. In the field of gear manufacture at present, design and manufacture process of tooth surface about spiral bevel gear are very complex and the control of quality is very difficult[2]. So spiral bevel gears are a research focus in gear manufacture field[3].

Accordance with the principle of gear engagement, tooth surface of spiral bevel gear should be tooth surface with spherical involute. However, the existing processing technology is performed by using engineering approximation method[4]. So there are theoretical errors on the tooth surfaces. First, transmission ratio is not constant; Second, modification and adjustment of tooth surfaces are complex; Finally, designing and manufacturing need to take a longer period and gears can not be interchanged unless in the same batch. Therefore, in order to reduce to effect of the error, the parameters of machine tools and cutting tool need to be modified and improved continuously[5].

The new method that is put forward can be used to process ideal spherical involute spiral bevel gears that the transmission ratio is constant and gears can be interchanged even in different batches. This method makes the calculation and adjustment of machine tools settings relatively simple, the processing efficient relatively high. It is proved to be effective and feasible by the experiments.

II. THE GENERATION PRINCIPLE AND MATHEMATICAL MODELING OF SKEW BEVEL GEARS

There is assumed a sphere which is setting radius dimension as R and a base cone which culmination and the sphere center coincide, as shown in figure 1. The base cone and the great circle of a sphere-a plane Q are tangent to a line. When the base cone is fixed, the plane Q rotate around the base cone and they keep tangency all along. Under the condition of the above expression, the movement of the plane Q relative to the base cone is what is needed pure roll for generating spherical involute. If there is a straight line ML on the plane Q. In the expansion, the straight line ML can spread a curved surface, while the curved surface is then involute helicoids.

As shown in figure 2. Base cone angle \( \delta \) generatrix of small end about base cone Rb base cone circle spreading angle \( \phi \) plane Q spreading angle \( \gamma \). Fixed coordinate systems \( S(o-x,y,z) \) are rigidly connected to the gear blanks. In this systems, origin \( o1 \)and \( x1-y1 \) plane coincide with center of cone and bottom of base cone respectively; At the same time, Matrix \( M1t \) provides the coordinate transformations from system \( S1 \) into the top coordinate system \( S(t-o,x,y,z) \) by offsetting \( A \) along the axis \( z1 \); And matrix \( Mbs \) performs the relation between the stationary coordinate systems \( St \) and \( Sb(ox,xs,ys,zs) \), in which coordinate systems \( Ss \) rotate \( \phi \) on the axis \( zt \); Afterward Matrix \( Msb \) performs the coordinate transformations from system \( Ss \) into the top coordinate system \( Sb(ox,xb,yb,zb) \) by moving \( \pi/2-\delta \) around the axis \( ys \), at the same time axis \( xb \) and origin \( ob \) coincide with the line of base cone and the plane Q respectively[6]; Finally, \( Mbq \) denote the transformations from \( Sb \) into \( Sq(qq-xq,yq,zq) \) by revollying \( \gamma \) around \( zb \) connected to the plane Q.
In the spreading process, a cluster of tracing line that are formed by generation line ML make up involute helicoids. In the coordinate systems S, the value of the generation line ML can be acquired by Coordinate Transform from coordinate systems $S_q$ [7]. When parameter $\phi$ continuous change, in the coordinates system $S(o-x,y,z)$ the mathematical model of a point on the Tooth-surface can be performed by applying the following equation:

$$
\begin{bmatrix}
    x \\
    y \\
    z
\end{bmatrix} = \begin{bmatrix}
    M_{bl} & M_{bn} & M_{bt} & M_{bt}
\end{bmatrix} \begin{bmatrix}
    x_b \\
    y_b \\
    z_b \\
    t_b
\end{bmatrix}
$$

Where,

$$
M_{bl} = \begin{bmatrix}
    \cos(\psi) & \sin(\psi) & 0 & 0 \\
    -\sin(\psi) & \cos(\psi) & 0 & 0 \\
    0 & 0 & 1 & 0 \\
    0 & 0 & 0 & 1
\end{bmatrix}
$$

$$
M_{bt} = \begin{bmatrix}
    \cos(\pi/2 - \delta_b) & 0 & \sin(\pi/2 - \delta_b) & 0 \\
    0 & 1 & 0 & 0 \\
    -\sin(\pi/2 - \delta_b) & 0 & \cos(\pi/2 - \delta_b) & 0 \\
    0 & 0 & 0 & 1
\end{bmatrix}
$$

$$
M_{bn} = \begin{bmatrix}
    \cos(\phi) & -\sin(\phi) & 0 & 0 \\
    \sin(\phi) & \cos(\phi) & 0 & 0 \\
    0 & 0 & 1 & 0 \\
    0 & 0 & 0 & 1
\end{bmatrix}
$$

When big circle(plane Q) pure roll, the circular arc that base cone bottom circle is bypassed by big circle is same length. In the cartesian coordinate system, the vector equation of tooth surface about circle spreading angle $\phi$ and polar Radius $\rho$ is shown:

$$
r^1 = x(\phi,\rho)i + y(\phi,\rho)j + z(\phi,\rho)k
$$

Figure 2. Coordinate Transform

III. ANALYSIS OF SKEW BEVEL GEARS CUTTING MOVEMENT

Based on the above expression, profile of gears surface is formed accurately in accord with surface generating principle by the milling cutters which processes gears along tracing of generating line ML that is also generation line, if the generation line is as chord of face circular milling cutters. The plane Q spread on the base cone is equal to that plane Q turn around a center with angular velocity $\omega(b)$ and rotate around axis of the base cone at the angle rate of $-\omega(t)$ simultaneity, namely, the plane Q relative to of the base cone move with angular velocity $\omega(b)-\omega(t)$. When the base cone rotate $\omega(t)$ around axis of itself, spreading movement of tooth surface established can be decomposed that: base cone rotates with angular velocity vector $\omega(t)$, at the same time generation line ML on the plane Q rotate around itself axis at the angle rate vector of $\omega(b)$ relative to space coordinate. So there is a relationship between them:

$$
\frac{\omega_b}{\omega_t} = \sin \delta_b
$$

Figure 3. Analysis of Cutting Movement

As shown in figure 3, according to above motion relationship, chord of face circular milling cutters rotate with angular velocity $\omega(b)$ following plane Q. When generation line is in line segment WV, the chord cut in small end of bevel gear. When the chord moves to W1, it shows that the cutter will cut into root cone, and the overcut will happen. Tool edge rotates following Q. Meanwhile, it moves along chord direction. So the motion direction of the tool edge is along line segment OW' in order to avoid overcut. Because circular milling cutters rotate with angular velocity $\omega(b)$ the velocity of perpendicular to line OW direction is:

$$
R_0 \cdot \sec \delta_b \cdot t \cdot \sigma_b
$$

Figure 4. Cutting Movement of Circular Milling Cutters
According to right triangle OGW, the velocity of OW direction is shown:

$$v_{ow} = \frac{R_m \cdot \sin \omega_b t}{\cos \omega_b t}$$  \hspace{1cm} (5)

As shown in figure 4, circular milling cutters which is selected feature simple structure. Main cutting edge of milling cutters is in the same plane and the plane is vertical to tool shaft. The main cutting edge in high speed rotation form rotary plane $\Sigma$, chord WV in which can be as generation line to realize motion of tooth surface generation.

IV. MACHINE TOOL DESIGN OF SPIRAL BEVEL GEARS

Initial position relation of chord of circular milling cutters and workpiece are determined, through adjust position of worktable X,Y direction and degree of Z direction as well as change angle of inclined rotary table C and level rotary table B based on base cone angle $\delta_b$. When spiral bevel gears are processed, the work-shaft and level rotary table rotate with angular velocity $\omega_t$ and $\omega_b$ respectively to satisfy the equation 4 that ensure the strict proportion of the tool shaft and the workpiece movement so as to realize tooth surface spread correctly. Movement of the tool shaft and workpiece are as above mentioned. Meanwhile, Y axis move with angular velocity based on equation 5 to avoid overcut and complete feed drive. Therefore, the relation of three Axes linkage is composed with work-shaft. Level rotary table rotation and Y axis motion to realize milling of skew bevel gears.

V. EXPERIMENTAL VERIFICATION OF MILLING AND DETECTION

In this milling experiment, the material of milling cutter designed is cemented carbide, cutter Diameter is 350mm, tooth number is 12, length of cutter tooth is 13mm, the rake angle of main cutting edge is 5°, Celief angle of pair cutting edge is 6°. The material of a pair skew bevel gears is aluminum alloy. Main parameter of pinion and gearwheel are given in table 1.

![Figure 5. 3D Model of Machine Tool](http://www.i-joe.org)

![Figure 6. Cutting Processing of Gearwheel](http://www.i-joe.org)

![Figure 7. Cutting Processing of Pinion](http://www.i-joe.org)

<table>
<thead>
<tr>
<th>Modeling parameters</th>
<th>pinion</th>
<th>gearwheel</th>
</tr>
</thead>
<tbody>
<tr>
<td>Tooth number $z$</td>
<td>14</td>
<td>65</td>
</tr>
<tr>
<td>Big end module $m$/mm</td>
<td>6</td>
<td>6</td>
</tr>
<tr>
<td>Root cone angle $\delta$/$^\circ$</td>
<td>11.2171</td>
<td>11.2171</td>
</tr>
<tr>
<td>Big end pitch circle diameter $D$/mm</td>
<td>84</td>
<td>390</td>
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<tr>
<td>Base cone spiral angle $\beta$/°</td>
<td>35</td>
<td>35</td>
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<tr>
<td>Pitch cone angle $\delta$/$^\circ$</td>
<td>12.1549</td>
<td>77.8451</td>
</tr>
<tr>
<td>Pressure angle $a$/$^\circ$</td>
<td>22.5</td>
<td>22.5</td>
</tr>
<tr>
<td>Root cone angle $\delta$/$^\circ$</td>
<td>11.2171</td>
<td>11.2171</td>
</tr>
<tr>
<td>Face cone angle $\delta$/$^\circ$</td>
<td>14.5833</td>
<td>78.6156</td>
</tr>
<tr>
<td>Base cone angle $\delta$/$^\circ$</td>
<td>11.2171</td>
<td>64.5777</td>
</tr>
<tr>
<td>Face width $b$/mm</td>
<td>60</td>
<td>60</td>
</tr>
<tr>
<td>Cutting area $\mu$/$^\circ$</td>
<td>9.3744</td>
<td>9.5027</td>
</tr>
<tr>
<td>Adjusting angle of cutting area $\psi$/$^\circ$</td>
<td>0</td>
<td>54.0892</td>
</tr>
<tr>
<td>Generatrix of big end about base cone $L_b$/mm</td>
<td>199.4992</td>
<td>117.1003</td>
</tr>
<tr>
<td>Radius of Tangent Circle about Toothed Portion $R_b$/mm</td>
<td>97.2207</td>
<td>49.9587</td>
</tr>
<tr>
<td>The Length of generation line $[C]$/$^\mu$m</td>
<td>52.2233</td>
<td>81.0328</td>
</tr>
<tr>
<td>Center distance of cutting edge $L_o$/mm</td>
<td>173.0410</td>
<td>170.245</td>
</tr>
</tbody>
</table>
Based on the mathematical model of tooth surface about skew bevel gears, accurate 3D point cloud model of bevel gears are built in matlab. Meanwhile, plan the measuring path on the ideal tooth surface and establish theories coordinate system that is in keeping with measuring-coordinate system. Then, the datum of mesh nodes on the actual tooth surface are collected by coordinate measuring machine, and so these datum of discrete points of actual tooth surface are introducted into matlab system. According to coordinate transformation, spatial position error of discrete points between actual tooth surface and ideal tooth surface is calculated. Finally, tooth surface error of skew bevel gears is evaluated by means of difference surface[9].

Given \((r+1)\times(s+1)\) data points \(Q_{k,l}, k=0...r, l=0...s\), non-uniform B-spline surface can be fitted to 3\times3th power and \((m+1)\times(n+1)\) points, namely:

\[
S(\phi, \rho) = \sum_{i=0}^{r} \sum_{j=0}^{s} N_{i,3}(\phi)N_{j,3}(\rho)P_{i,j}
\]  

Where, \(N_{i,3}(\phi)N_{j,3}(\rho)\) is B-spline basis function, \(P_{i,j}\) is control points. Derivative of spline curve can get to be known that, partial derivative along \(u\) direction and \(v\) direction respectively are needed in order to get the surface normal. Partial derivative of \(u\) direction:

\[
S_u(\phi, \rho) = \sum_{i=0}^{r} \sum_{j=0}^{s} N_{i-1,2}(\phi)N_{j,3}(\rho)P_{i,j}^{(0,1)}
\]  

Where, \(P_{i,j}^{(0,1)} = 3 \Phi_{i+1} - 3 \Phi_{i+1} - \Phi_{i+1}\), knot vector:

\(\phi^{(0)} = \{0, 1, 2, \cdots, 0, 1, 2\}\) \(\rho^{(0)} = 0\). In the same way, partial derivative of \(u\) direction:

\[
S_v(\phi, \rho) = \sum_{i=0}^{r} \sum_{j=0}^{s} N_{i,3}(\phi)N_{j-1,2}(\rho)P_{i,j}^{(1,0)}
\]  

\(S_u(\phi, \rho)\) and \(S_v(\phi, \rho)\) are tangent vector of curved surface along \(u\) and \(v\) direction respectively. Because normal vector of one point on curved surface is perpendicular to tangent vector of any direction by the point, the normal vector equal to cross product of two cross product as follows:

\[
S_{\omega}(\phi, \rho) = S_u(\phi, \rho) \times S_v(\phi, \rho)
\]  

Measuring probe of 3D coordinate measuring machine presets a certain position of initial measuring point in normal direction on the basis of the vector formula 12. Measuring element mesh nodes on convex and concave of spiral bevel gears are measured in normal direction. The datum of all tooth in turn collecte average a set of numbers[10].

The maximum positive deviation about convex of gearwheel is less than 0.0379mm, maximum negative deviation while 0.0491mm. Similarly, the maximum positive deviation about concave of gearwheel is less than 0.0289mm, maximum negative deviation while 0.0390mm. The trend of error: convex from small end to big end shown an increasing tendency, maximum deviation is present to top of small end. Concave from small end to big end shown an increasing tendency.
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The maximum positive deviation about convex of pinion is less than 0.0370mm, maximum negative deviation while 0.0482 mm. Similarly, the maximum positive deviation about concave of pinion is less than 0.0467mm, maximum negative deviation while 0.0358 mm. The trend of error: convex from small end to big end showed an increasing tendency, maximum deviation is present to top of small end. Concave from small end to big end showed an increasing tendency, maximum deviation is present to top of small end.

VI. CONCLUSIONS

Generation line cutting methods is used for skew bevel gears. Namely, the generation line is as chord of face circular milling cutters, the tracing that generation line spread involute helicoids is just the cutting trace chord of face circular milling cutters. Skew bevel gears without theoretic errors are machined using three axes linkage by this method. The cutting motion and control is greatly simplified by utilizing this method. Machining experiments of a pair of mesh gear are performed by generation line processing method. And then, the measurement and evaluation of the pairs are completed. Detection results :the maximum positive deviation about convex of gearwheel is less than 0.0491mm. The maximum positive deviation about convex of pinion is less than 0.0482mm. The experimental results show that this method is feasible and effective.

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